

Performance and Emissions Characteristics of Variable Compression Ignition Engine

K Satyanarayana¹, Naik RT^{2*} and SV Uma-Maheswara Rao³

¹ANITS, Mechanical Engineering, Andhra University, Visakhapatnam -531162, India

²Department of Mechanical Engineering, Indian Institute of Science, Bangalore-560012, India

³Department of Marine Engineering, Andhra University, Visakhapatnam-531006, India

Abstract

Variable Compression Ratio engine test rig is used to determine the effect of compression Ratio on the performance and emissions of the engine. The objective is to determine the optimum compression ratio for the better performance and lowering the emissions. In order to determine the optimum compression ratio, experiments were carried out on a single cylinder four stroke variable compression ratios of 16.5, 17.0, 17.5, 18.0 and 19.0 in a Diesel engine. The performance characteristics of the engine namely break power, brake thermal efficiency, brake specific fuel consumption and other parameters are studied and also conducted the emissions test namely Carbon monoxide, carbon dioxide, hydrocarbons, Nitrogen oxides and other emissions at various conditions. It is observed that a significant improvement in the performances and lowered the various emissions also at a compression ratio of 17.0 compared to other compression ratios due to the enhancement of the overall combustion process.

Keywords: Performance emissions; Variable compression ratio; Thermal efficiency; Specific fuel consumption engine

Introduction

The ultimate goal of emission legislation is to force technology to the point where a practically viable zero emission vehicle become a reality with formidable challenges [1]. The ever increasing demand for the conventional based fuels and their scarcity has led to extensive research on Diesel fuelled engines [2]. A better design of the engine can significantly improve the combustion quality and in turn may lead to better break thermal efficiency and hence saves fuel [3]. India though rich in coal abundantly and endowed with renewable energy in the form of solar, wind, hydro and bio-energy has a very small hydro carbon reserves, 0.4% of the world's reserve[4]. India is a net importer of energy and nearly 25% of its energy needs are met through imports mainly in the form of crude oil and natural gas [5]. The rising oil bill has been the focus of serious concerns due to the pressure it has placed on scarce foreign exchange resources and is also largely responsible for energy supply shortages [6]. The sub-optimal consumption of commercial energy adversely affects the productive sectors, which in turn hampers economic growth [7]. All over the world, reduction of automotive fuel consumption and CO₂ emissions is leading to the introduction of various new technologies in engines [8].

Diesel engine

The diesel engine is an internal combustion engine in which ignition of the fuel that has been injected into the combustion chamber is initiated by the high temperature which a gas achieves when greatly compressed [9]. This contrasts with spark-ignition engines such as a gasoline engine or gas engine, which use a spark plug to ignite an air-fuel mixture [10]. The diesel engine has the highest thermal efficiency of any standard internal or external combustion engine due to its very high compression ratio and inherent lean burn which enables heat dissipation by the excess air [11]. A small efficiency loss is also avoided compared to two-stroke non-direct -injection gasoline engines since un burnt fuel is not present at valve overlap and therefore no fuel goes directly from the intake/injection to the exhaust [12]. Low-speed diesel engines as used in ships and other applications where overall engine weight is relatively unimportant can have a higher thermal efficiency [13].

Variable compression ratio

Variable compression ratio (VCR) is technology to adjust the

compression ratio of an internal combustion engine while the engine is in operation. This is done to increase fuel efficiency while under varying loads [14]. Higher loads require lower ratios to be more efficient and vice versa. Variable compression engines allow for the volume above the piston at 'Top dead centre' to be changed. For automotive use this needs to be done dynamically in response to the load and driving demands. Especially, gasoline engines have a limit on the maximum pressure during the compression stroke, after which the fuel/air mixture detonates rather than burns [15]. To achieve higher power outputs at the same speed, more fuel must be burned and therefore more air is needed [16]. To achieve this, turbochargers or superchargers are used to increase the inlet pressure. This would result in detonation of the fuel/air mixture at higher compression ratios [17]. However, optimal VCR expected to improve the performance of the Compression Ignition Engine at various conditions.

Experimental Setup

The VCR engine chosen to carryout experimentation is a single cylinder, water cooled, vertical, direct injection, constant speed, CI engine. This engine can with stand higher pressures encountered and also used extensively in agricultural and industrial sectors [18-22]. Therefore this engine is selected for conducting experiments. Moreover necessary modifications on the piston and the cylinder head can easily be made. The engine has a eddy current dynamometer to measure its output. it consists of stator on which are fitted with a number of electromagnets and a rotor disc made of a copper or steel and coupled to shaft of engine . When the rotor rotates eddy currents are produced in the stator due to magnetic flux set up by the passage of field current in the electro magnets. This eddy current opposes the rotor motion thus loading the engine (Figure 1).

***Corresponding author:** Naik RT, Department of Mechanical Engineering, Indian Institute of Science, Bangalore-560012, India, Tel: +91-80-22932960; E-mail: rt naik@mecheng.iisc.ernet.in.

Received June 01, 2016; **Accepted** July 05, 2016; **Published** July 08, 2016

Citation: Satyanarayana K, Naik RT, Uma-Maheswara Rao SV (2016) Performance and Emissions Characteristics of Variable Compression Ignition Engine. Adv Automob Eng 5: 146. doi:10.4172/2167-7670.1000146

Copyright: © 2016 Satyanarayana K, et al. This is an open-access article distributed under the terms of the Creative Commons Attribution License, which permits unrestricted use, distribution, and reproduction in any medium, provided the original author and source are credited.

Performance of the engine

Brake specific fuel consumption (BSFC): Brake Specific fuel consumption (BSFC) or sometimes simply Brake specific fuel consumption, BSFC, is an engineering term that is used to describe the fuel efficiency of an engine design with respect to thrust output. Brake Specific Fuel Consumption may also be thought of as fuel consumption generally in grams/sec.

Brake thermal efficiency: Brake Thermal Efficiency is defined as brake power of a heat engine as a function of the thermal input from the fuel. It is used to evaluate how well an engine converts the heat from a fuel to mechanical energy.



Figure 1: Experimental setup of VCR diesel engine.



Figure 2: Smoke meter.

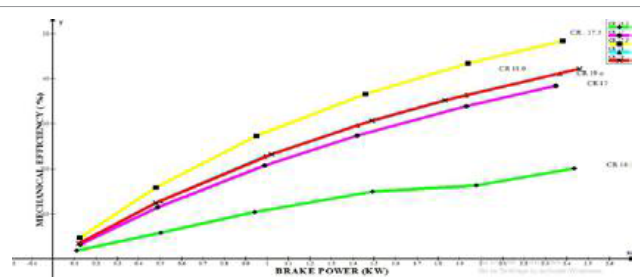


Figure 3: Effect of brake power with mechanical efficiency.

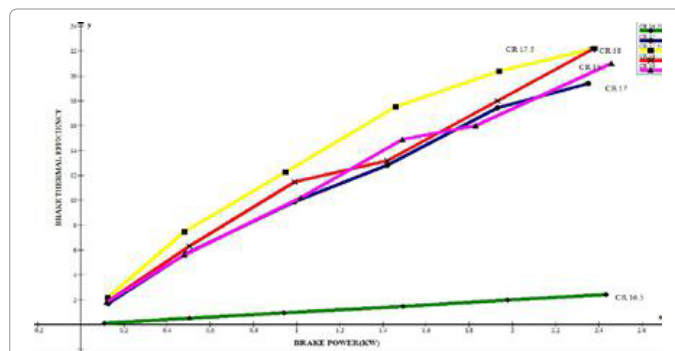


Figure 4: Effect of brake power with brake thermal efficiency.

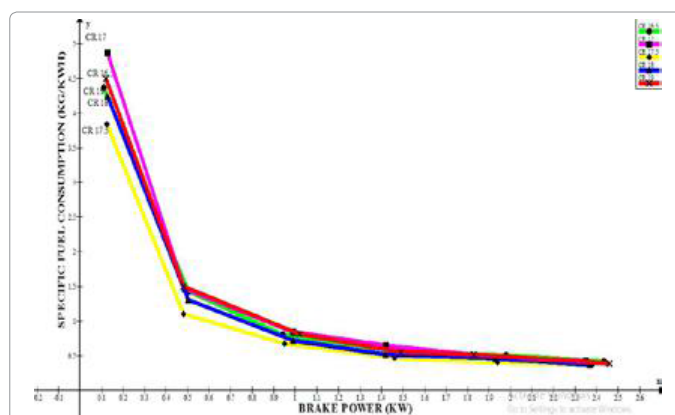


Figure 5: Effect of brake power with specific fuel consumption.

Emission gas analyser

Exhaust Gas Analyzer is a non-dispersive infrared Fuji gas analyzer is used to measure the various emissions namely CO, CO₂, HC, NO_x, O₂ and smoke test also carried out with smoke meter as shown in Figure 2.

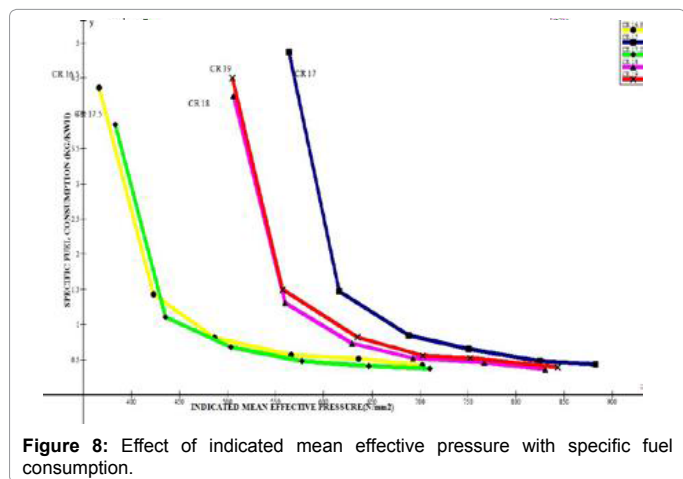
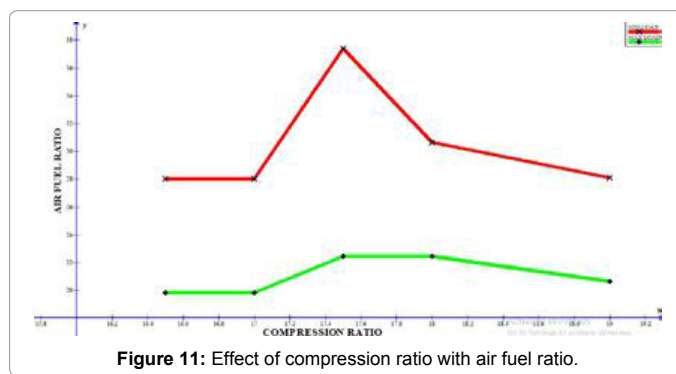
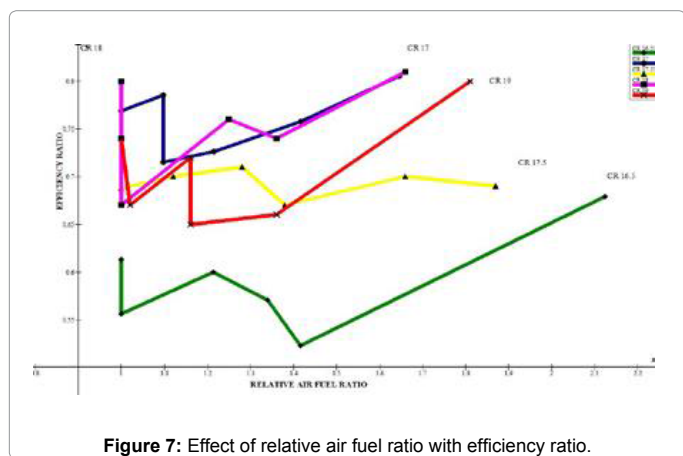
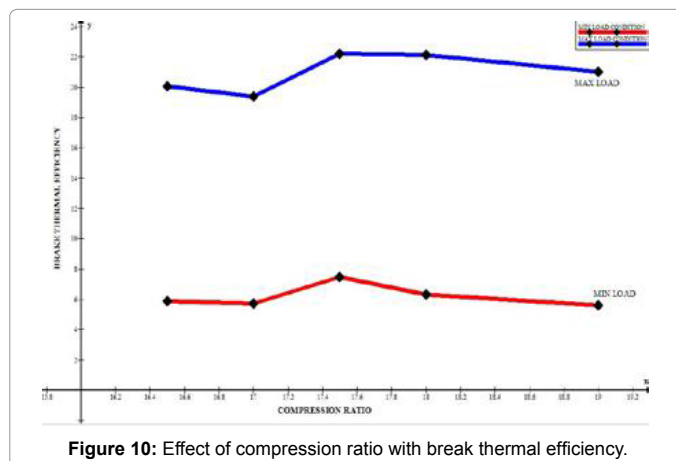
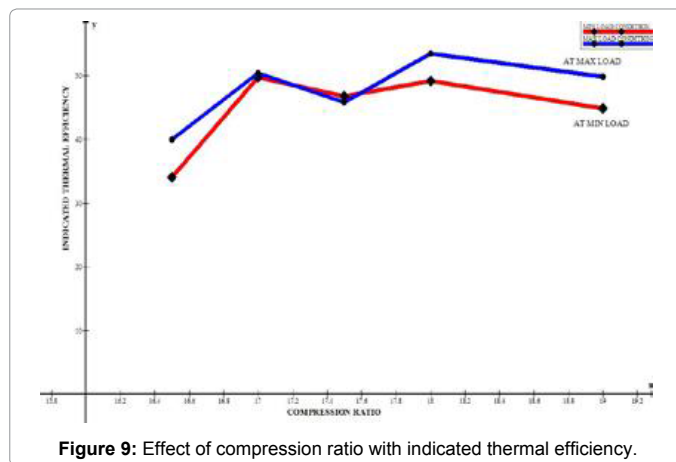
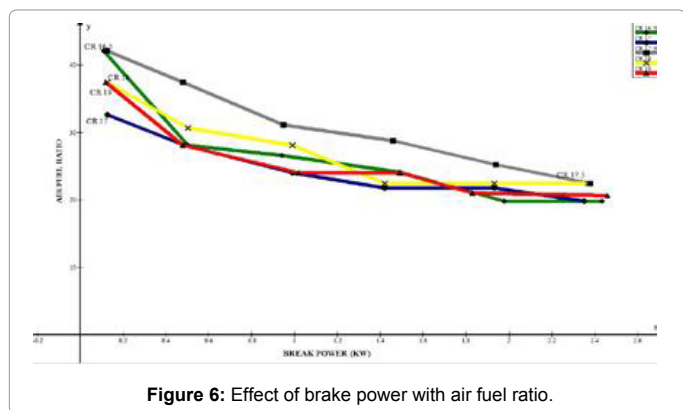
Results and Discussion

The various performance and emissions of the VCR engine with different compression ratios discussed.

Performance analysis

The variation in mechanical efficiency at different loads for different compression ratios is shown in Figure 3. It is observed that mechanical efficiency increases with the increase of the load. It is observed that mechanical efficiency is low for compression ratio of 16.5 and it is almost similar trend for 18 and 19 for a given load. But, mechanical efficiency is higher for compression ratio of 17 compared to other compression ratios due to efficient combustion (Figure 4). The variation in brake thermal efficiency at different loads for different compression ratios. It is observed that brake thermal efficiency is lower for compression ratio of 16.5, it is almost similar for all other compression ratios at all loads. However, thermal efficiency is more for compression ratio of 17 compared to other compression ratios due to proper combustion (Figure 5). The variation in Specific fuel consumption at different loads for different compression ratios is shown in Figure 5. It is observed that at higher loads the Specific fuel consumption of all the compressions ratios are almost at higher break power. But, fuel consumption is lower for the compression ratio of 17 compared to other compression ratios due to the proper combustion. The variation

in Air fuel ratio at different loads for different compression ratios is shown in (Figure 6). It is observed that air fuel ratio is almost similar pattern for all compression ratios at higher loads But, at lower loads of air fuel ratios is lower for the compression ratios of 17 compared to other compression ratios due to efficient combustion (Figure 7). The variation between relative air fuel ratio and efficiency ratio. It is observed that efficiency ratio is less for the compression ratio of 16.5 and it increases with other compression ratios also. But, maximum efficiency ratios observed for the compression ratios of 17 compared to other compression ratios due to proper air fuel mixture and efficient



combustion (Figure 8). The variation of specific fuel consumption with indicated mean effective pressure is shown in Figure 8. It is observed that specific fuel consumption is almost equal for all the compression ratios at higher indicated pressure due to smooth combustion. The variation between compression ratio and indicated thermal efficiency is shown in (Figure 9). It is observed that for a given compression ratio indicated thermal efficiency is higher for maximum load and lower for minimum loads due to proper combustion. The variation between compression ratio and break thermal efficiency is shown in Figure 10. It is observed that for a given compression ratio break thermal efficiency is higher for maximum load and lower for minimum load due to efficient combustion. The variation between compression ratio and air fuel ratio is shown in Figure 11. It is observed that for a given

compression ratio air fuel ratio is higher for maximum load and lower for minimum load due to proper combustion (Figure 12).

Emission analysis

The variation of percentage of Carbon Monoxide (CO) with the air fuel ratio is shown in Fig.12. CO emissions are reduced by lowering the compression ratios. But, CO emission is lower for the compression ratio of 17 due to efficient combustion compared to other compression ratios. The variation of Carbon Dioxide (CO₂) with the variation of air fuel ratio is shown in Figure 13. It is observed that at lower loads, CO₂ is similar trend for all other compression ratios. But, CO₂ is lower for the compression ratio of 17 due to efficient combustion compared to other compression ratios (Figure 14). The variation of Oxygen (O₂) with the

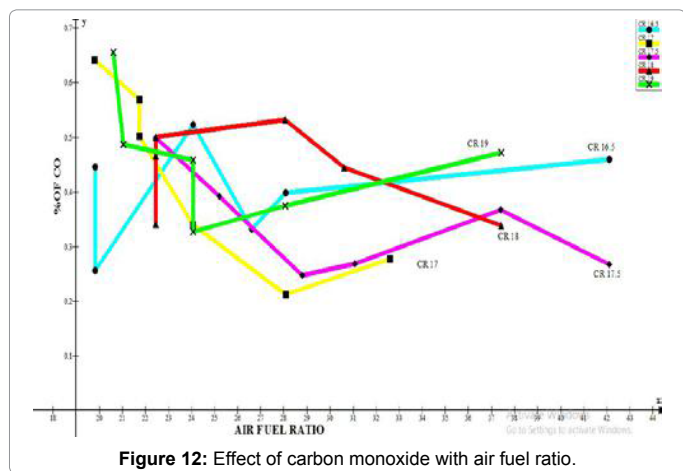


Figure 12: Effect of carbon monoxide with air fuel ratio.

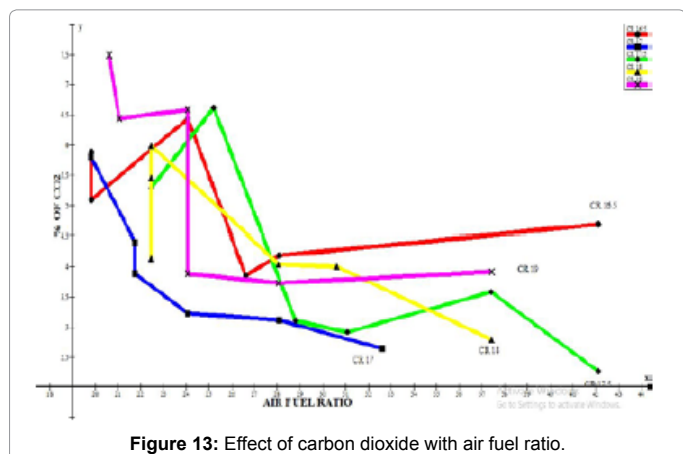


Figure 13: Effect of carbon dioxide with air fuel ratio.

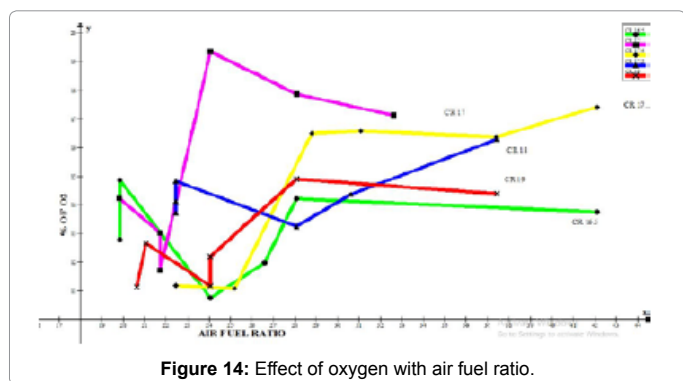


Figure 14: Effect of oxygen with air fuel ratio.

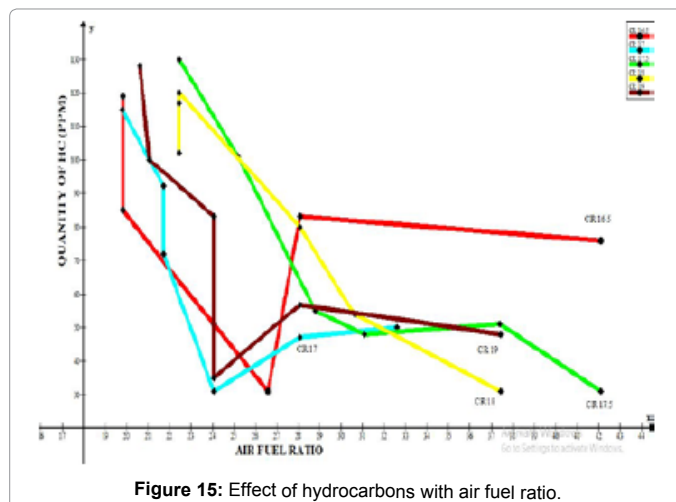


Figure 15: Effect of hydrocarbons with air fuel ratio.

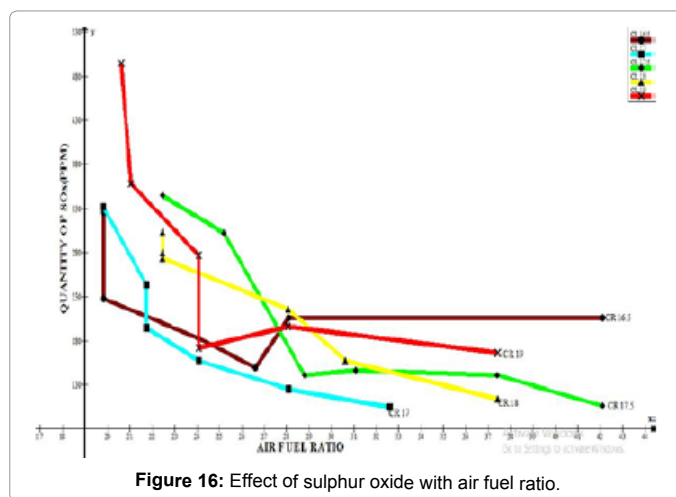


Figure 16: Effect of sulphur oxide with air fuel ratio.

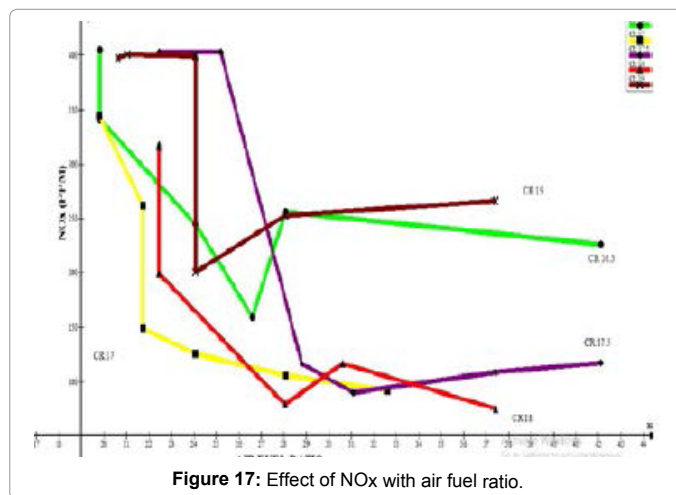


Figure 17: Effect of NOx with air fuel ratio.

air fuel ratio. It is observed that oxygen is lower for all the compression ratios. But, Oxygen is higher for the Compression ratio of 17 compared to other compression ratios which indicated the better and smooth combustion. The variation of unburned hydrocarbon (HC) with air fuel ratio shown in Figure 15. The emissions are seen to be lower at compression ratio of 17 compared to other compression ratios due to efficient combustion at higher loads. (Figure 16). The variation of

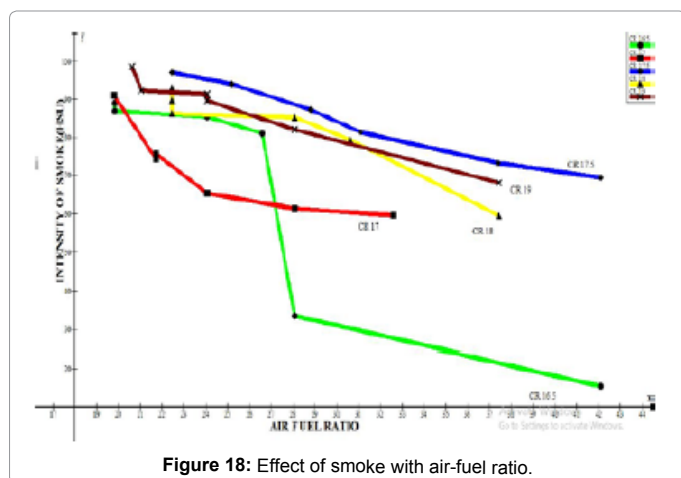


Figure 18: Effect of smoke with air-fuel ratio.

Sulphur Oxide (SO_x) with air fuel ratio is shown in It is observed that SO_x is similar trend for higher compression ratios at higher loads. But, SO_x is lower for the compression ratio of 17 compared to other compression ratios due to efficient combustion. The variation of NO_x emission with air fuel ratio is shown in Figure 17. It is observed that NO_x is increasing with similar pattern for all the compression ratios due to increase of cylinder temperature. But, NO_x is still lower for the compression ratios of 17 compared to other compression ratios due to proper combustion. The variation of intensity of smoke with the variation of air fuel ratio is shown in Figure 18. It is observed that smoke is decreasing for all the compression ratios at higher loads due to efficient combustion.

Conclusions

1. The Break thermal efficiency, Mechanical efficiency and efficiency ratios are higher for the compression ratio of 17 compared to other compression ratios due to efficient combustion.

2. Break Specific fuel consumption is lower and Break Power is almost similar with minor variations for the compression ratio of 17 compared to other compression ratios due to enhanced combustion.

3. Indicated mean effective pressure is higher for the compression ratio of 17 compared to other compression ratios due to efficient combustion.

4. The various emissions namely CO, CO₂, HC, SO_x and O₂ decreasing for the compression ratio of 17 compared to other compression ratios due to efficient combustion.

5. The NO_x emission increasing for all the compression ratios due to increase of cylinder temperature. However, still NO_x is lower in the case of 17, compression ratio due to proper combustion.

6. The Smoke emission is also lower for all the compression ratios at higher loads due to enhanced combustion.

Acknowledgement

Authors express their sincere gratitude to their teacher Prof. P. V. Krishnan for his wonderful training and teachings.

References

- Heywood Internal Combustion Engine Fundamentals (1988) New York: McGraw Hill.
- Yoon M, Kim S, Hyun B, Woo R, Sik LC et al. (2008) Combustion and emission characteristics of DME as an alternative fuel for compression ignition engines with a high pressure injection system *Fuel*. 87: 2779-2786.
- Zhu R, Wang X, Miao H, Huang Z, Gao J (2008) Performance and emission characteristics of diesel engines fuelled with diesel dimethoxymethane blends. *Fuel*. 87: 2779-2786.
- Devaradjane G (2008) Experimental investigation on performance and emission characteristics of diesel. Fuel blended with 2-Ethoxy Ethyle Acetate and 2-Butoxy Ethanol SAE Paper.
- Murugesan A, Umarani C, Subramanian R, Nedunchezian N (2009) Bio-diesel as an alternative fuel for diesel engines. A review *Renewable and Sustainable Energy Reviews*, pp: 653-662.
- Jindal S, Nandwana BP (2010) Investigation of the effect of compression ratio and injection pressure in a direct injection diesel engine running on Jatrophamethyl ester. *Applied thermal engineering*, pp: 442-448.
- Appa Rao BV (2011) Investigation on emission characteristics of diesel engine come-tricetion additive blends fuel. *International Journal of Advanced Engineering Research and Studies*. 1: 217-221.
- Sejal NP (2012) An experimental analysis of diesel engine using bio-fuel by varying compression ratio. *Journal of Engineering Sciences*, Vol-2.
- Prashant G (2011) Comparative study of hemp and jatropha oil blends used as an alternative fuel in diesel engine. *September CIGR Journal*, Vol-13.
- Sateesh Y (2013) Performance emission analysis of diesel engine using oxygenated compounds. *International Journal of Advances in Science and Technology*, 61: 9-16.
- Naik RT, Nilesh C (2016) Emission characteristic of a high speed diesel engine. *International Journal of Mechanical Engineering*, 5: 29-36.
- Davis N, Lents J, Osses M, Nikkila N, Barth M et al. (2005) Development and application of an international vehicle emissions model. *Transportation Research Board Annual Meeting Washington*.
- Solomon S (2007) *Climate changes: The physical science basis IPCC Fourth Assessment Report (AR4)*. Cambridge, United Kingdom and New York, USA: Cambridge University Press, pp: 996.
- Srinivasan S, Rogers P (2005) Travel behaviour of low-income residents: studying two contrasting locations in the city of Chennai. *India Journal of Transport Geography*, pp: 265-274.
- Subramanian KP (2008) *Public transportation system in Chennai, India: a review Municipalise-making cities work better Mumbai, India*.
- Kruse RE, Huls TA (1973) *Development for the federal urban driving cycle SAE Paper*.
- Kuhler M, Karstens D (1978) *Improved driving cycle for testing automotive exhaust emissions. SAE Technical Paper Series*.
- Lyons TJ, Kenworthy JR, Austin PI, Newman PWG (1986) *The development of a driving cycle for fuel consumption and emissions evaluation. Transportation Research*. pp: 447-462.
- Goto Y, Nurusawa K (1996) *Combustion of a spark ignition natural gas engine. Journal of SAE Review*, 17: 251-258.
- Moffat RJ (1988) *Describing the uncertainties in experimental results. Experimental Thermal Fluid Science*. 1: 3-17.
- Karim GA, Liu Z (1995) *The ignition delay period of dual fuel engines. SAE*.
- Selim MYE (2000) *Pressure-time characteristics of diesel engine fuelled with natural gas. Renew Energy Journal* 22: 473-489.