

A Dimensionless Analysis of Stress Distribution for Hydrostatic Tension in an Orthotropic Plate

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Abstract

The effect of lamina material properties on the stress distribution of the orthotropic composite plate containing a circular cutout under hydrostatic tension is investigated. Based on the generalized Hooke's law, the generalized plane stress condition and the complex variable method, a dimensionless analysis is used to evaluate the influence of various elastic moduli E_1 , E_2 , G_{12} and ν_{12} on the stress distribution along the boundary of the cutout in the composite plate under hydrostatic tension. The results obtained from this dimensionless analysis provide a set of general design guidelines for structural laminates with high precision requirements in the engineering applications.

Keywords: Hydrostatic tension; Stress distribution; Orthotropic plate; Complex variable; Composite materials

Introduction

Composite materials consist of various fibrous reinforcements coupled with a compatible matrix to achieve superior structural performance. The selection of composite materials for specific applications is generally determined by the physical and mechanical properties of the materials, evaluated for both function and fabrication [1]. Knowledge of stress distributions in anisotropic materials is very important for proper use of these new high-performance materials in structural applications [2].

Kuwamura [3] analyzed the stresses around the circular cutout by means of Ikeda's formula in consideration of orthotropic elasticity of the woods, which revealed that the stresses distribution around the circular hole is nearly equal to that of an isotropic plate. Selivanov [4] studied the time variation in the stresses around an elliptic hole in a composite plate. Kumar et al. [5] presented an approximate solution in the form of a polynomial for the normal stress distribution adjacent to a class of optimum holes in symmetrically laminated infinite composite plates under uniaxial loading. Giare and Shabahang [6] used a finite element analysis to calculate the stress distribution around a hole in a finite isotropic plate reinforced by composite material. Tsai et al. [7] developed a novel procedure for predicting the notched strengths of composite plates each with a center hole. And the stress distribution of a composite plate with a center hole is obtained by a finite element analysis.

In this paper, the effect of lamina material properties on the stress distribution of the orthotropic composite plate containing a circular cutout under hydrostatic tension is investigated. Based on the generalized Hooke's law, the generalized plane stress in an orthotropic composite plate and the complex variable method, a dimensionless analysis is used to evaluate the influence of various elastic moduli E_1 , E_2 , G_{12} and ν_{12} on the stress distribution along the boundary of the circular cutout in the composite plate under hydrostatic tension.

Tangential Stresses on the Boundary of the Circular Opening

The stress-strain distribution of an infinite anisotropic plate containing a through-the-thickness cutout has been derived using a complex variable method [8-10]. For an orthotropic plate subjected to

equal tensile stress p in two principal directions which are applied at a considerable distance from the circular opening as shown in Figure 1 (and this is equivalent to hydrostatic tension in plane xy). The normal stress component σ_θ for an element tangential to the opening is [9]

$$\sigma_\theta = p \frac{E_0}{E_1} [-k + k(k+n)\cos^2\theta + (1+n)\sin^2\theta] \quad (1)$$

where

$$k = \sqrt{\frac{E_1}{E_2}} \quad (2)$$

$$n = \sqrt{\frac{E_1}{G_{12}} - 2\nu_{12} + 2\sqrt{\frac{E_1}{E_2}}} \quad (3)$$

The θ is the polar angle measured from the x -axis; E_0 is the Young's modulus in tension (compression) along the direction tangent to the opening contour, which is related to elastic constants in the principal directions by the formula

$$\frac{1}{E_0} = \frac{\sin^4\theta}{E_1} + \left(\frac{1}{G_{12}} - \frac{2\nu_{12}}{E_1}\right)\sin^2\theta\cos^2\theta + \frac{\cos^4\theta}{E_2} \quad (4)$$

For an isotropic plate [9]

$$\sigma_\theta = 2p \quad (5)$$

A Dimensionless Analysis Model for Evaluation of the Normal Stress σ_θ

Consider the orthotropic plate containing a circular hole under hydrostatic tension. For a dimensionless analysis, the three lamina

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Received May 21, 2014; Accepted July 05, 2014; Published July 11, 2014

Citation: Yeh HL, Yeh HY (2014) A Dimensionless Analysis of Stress Distribution for Hydrostatic Tension in an Orthotropic Plate. J Aeronaut Aerospace Eng 3: 129. doi:10.4172/2168-9792.1000129

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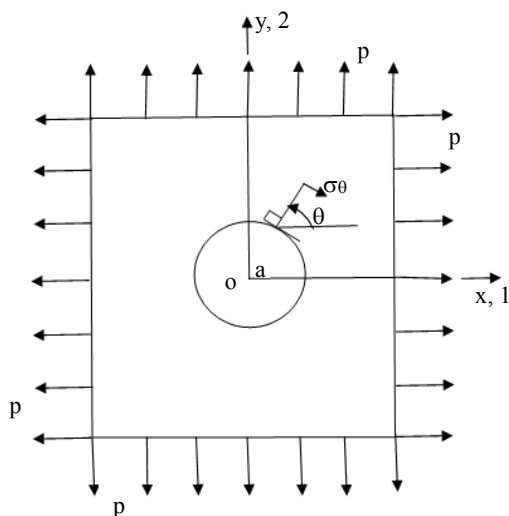


Figure 1: An orthotropic plate containing a circular hole under hydrostatic tension with principal material axes 1-2 and x-y axes.

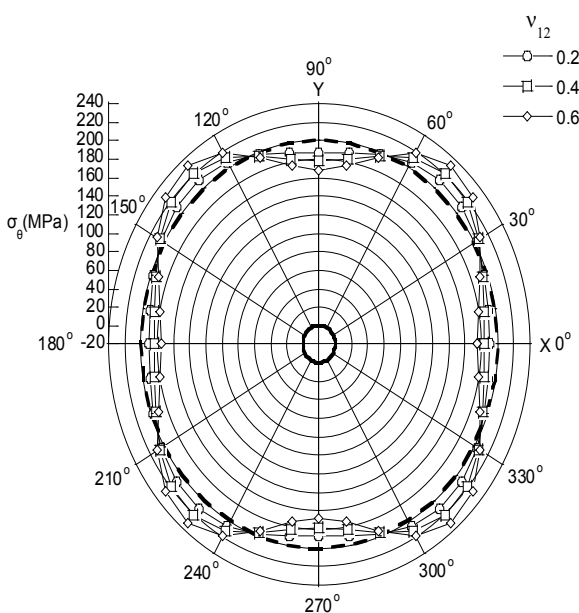


Figure 2: The normal stress component σ_θ vs. θ and ν_{12} for $p=100\text{MPa}$, $E_1=204\text{GPa}$, $E_2/E_1=0.6$, and $G_{12}/E_1=0.4$.

material constants E_1 , E_2 and G_{12} are represented by two dimensionless ratios E_2/E_1 and G_{12}/E_1 . Totally, the lamina material has three parameters E_2/E_1 , G_{12}/E_1 and ν_{12} . For an unidirectional lamina with different values of E_2/E_1 , G_{12}/E_1 and ν_{12} , the normal stress component σ_θ of the orthotropic plate under the hydrostatic tension will be varied and provided different values.

Various cases of different combinations of the three material parameters are considered in this study. In general case, the ranges of E_2/E_1 and G_{12}/E_1 are between zero and one, the ranges of ν_{12} are between 0 and 0.6. For a lamina with given material properties E_2/E_1 , G_{12}/E_1 and ν_{12} , the normal stress component σ_θ of the orthotropic plate under the hydrostatic tension can be evaluated.

Results and Discussion

In the (Figures 2-4), the bold solid line represents the circular hole and the bold dotted line shows the stress distribution σ_θ in an isotropic plate subjected to identical load as in an orthotropic plate. The stress distribution for an orthotropic plate is symmetrical with respect to the opening center. Thus, the variation of the stress distribution σ_θ is considered only in the range of $0^\circ \leq \theta \leq 180^\circ$ for an orthotropic plate. In an isotropic plate given $p=100\text{ MPa}$, then calculated from Equation (5), the constant stress σ_θ is 200 MPa. Therefore, the stress concentration factor in an isotropic plate is $S_c=2.0$. It is well known that the stress concentration factor for a circular cutout in the isotropic thin plate under uni-axial tensile load is “three”. However, in this study, tensile stresses in two perpendicular directions equal to p are applied in the thin plate with a circular cutout, therefore, through the linear superposition, it is found the stress concentration factor for this isotropic plate is $S_c=2.0$.

In Table 1 given $p=100\text{ MPa}$, $E_1=204\text{ GPa}$, $E_2/E_1=0.6$ and $G_{12}/E_1=0.4$, the variation of the stress σ_θ with respect to the polar angle θ and ν_{12} is the following:

With $\nu_{12}=0.2, 0.4$, and 0.6 , the minimum corresponding stresses σ_θ are 187.28 MPa, 177.83 MPa, and 167.93 MPa at $\theta=90^\circ$ respectively, and the maximum stresses σ_θ are 211.62 MPa, 221.17 MPa, and 232.14 MPa at $\theta=50^\circ$ as well as $\theta=130^\circ$ respectively. Also, the stress concentration factors are $S_c=2.12, 2.21$, and 2.32 respectively, for orthotropic case.

Table 1 shows that within the ranges of the polar angle $0^\circ \leq \theta \leq 20^\circ$, $70^\circ \leq \theta \leq 110^\circ$, and $160^\circ \leq \theta \leq 180^\circ$, the stress σ_θ decreases along with the increased values of ν_{12} . But, within the ranges of the polar angle $30^\circ \leq \theta \leq 60^\circ$ and $120^\circ \leq \theta \leq 150^\circ$, the stress σ_θ increases along with the increased values of ν_{12} . A summary of Table 1 is shown in Figure 2.

In Table 2 given $p=100\text{ MPa}$, $E_1=204\text{ GPa}$, $E_2/E_1=0.6$ and $\nu_{12}=0.4$, the variation of the stress σ_θ with respect to the polar angle θ and G_{12}/E_1 is following:

With $G_{12}/E_1=0.2, 0.4, 0.6, 0.8$ and 1.0 , the minimum corresponding

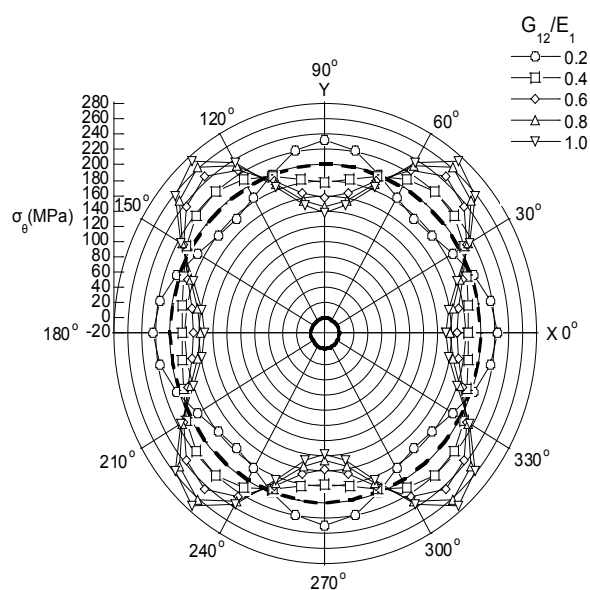


Figure 3: The normal stress component σ_θ vs. θ and G_{12}/E_1 for $p=100\text{ MPa}$, $E_1=204\text{ GPa}$, $E_2/E_1=0.6$, and $\nu_{12}=0.4$.

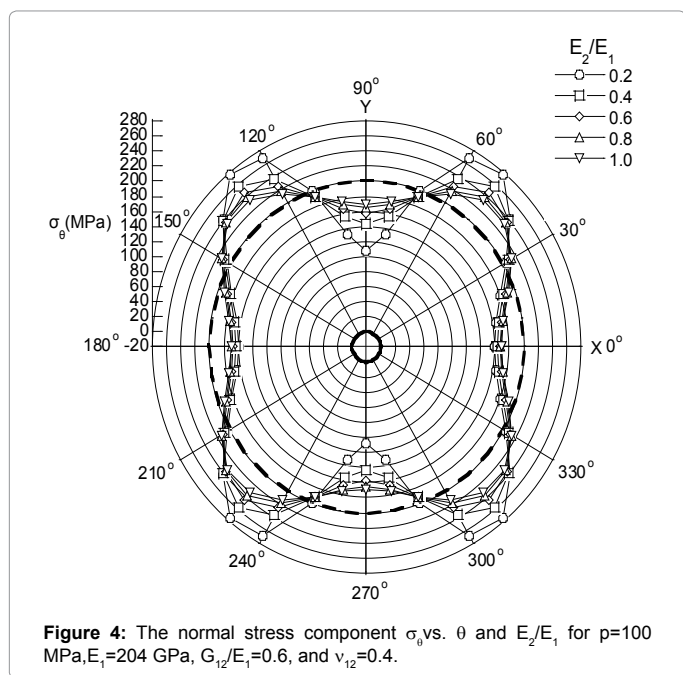


Figure 4: The normal stress component σ_θ vs. θ and E_2/E_1 for $p=100$ MPa, $E_1=204$ GPa, $G_{12}/E_1=0.6$, and $\nu_{12}=0.4$.

| θ | σ_θ (MPa) | | | |
|----------|-----------------------|----------|----------|----------|
| | ν_{12} | 0.2 | 0.4 | 0.6 |
| 0° | | 190.1468 | 182.8274 | 175.1573 |
| 10° | | 192.0270 | 185.9280 | 179.3790 |
| 20° | | 197.2014 | 194.6964 | 191.6553 |
| 30° | | 204.1280 | 207.0080 | 209.7919 |
| 40° | | 210.0384 | 218.0841 | 227.1022 |
| 50° | | 211.6244 | 221.1711 | 232.1421 |
| 60° | | 207.1710 | 212.6519 | 218.5025 |
| 70° | | 198.5809 | 196.9797 | 194.7377 |
| 80° | | 190.5158 | 183.1520 | 175.1465 |
| 90° | | 187.2796 | 177.8302 | 167.9282 |
| 100° | | 190.5158 | 183.1520 | 175.1465 |
| 110° | | 198.5809 | 196.9797 | 194.7377 |
| 120° | | 207.1710 | 212.6519 | 218.5025 |
| 130° | | 211.6244 | 221.1711 | 232.1421 |
| 140° | | 210.0384 | 218.0841 | 227.1022 |
| 150° | | 204.1280 | 207.0080 | 209.7919 |
| 160° | | 197.2014 | 194.6964 | 191.6553 |
| 170° | | 192.0270 | 185.9280 | 179.3790 |
| 180° | | 190.1468 | 182.8274 | 175.1573 |

Table 1: The normal stress component σ_θ vs. θ and ν_{12} for $p=100$ MPa, $E_1=204$ GPa, $E_2/E_1=0.6$, and $G_{12}/E_1=0.4$.

stresses σ_θ are 176.16 MPa at $\theta=50^\circ$ as well as $\theta=130^\circ$, and other minimum corresponding stresses σ_θ are 177.83 MPa, 156.61 MPa, 145.03 MPa and 137.69 MPa at $\theta=90^\circ$ respectively, and the maximum stresses σ_θ are 231.32 MPa at $\theta=90^\circ$, 221.17 MPa, 246.09 MPa, 262.21 MPa and 273.54 MPa at $\theta=50^\circ$ as well as $\theta=130^\circ$ respectively. Also, the stress concentration factors are $S_c=2.31, 2.21, 2.46, 2.62,$ and 2.73 respectively, for orthotropic case.

Table 2 shows that within the ranges of the polar angle $0^\circ \leq \theta \leq 20^\circ, 70^\circ \leq \theta \leq 110^\circ$ and $160^\circ \leq \theta \leq 180^\circ$ the stress σ decreases along with the increased values of G_{12}/E_1 . But, within the ranges of the polar angle $30^\circ \leq \theta \leq 60^\circ$ and $120^\circ \leq \theta \leq 150^\circ$, the stress σ_θ increases along with the increased values of G_{12}/E_1 . A summary of Table 2 is shown in Figure 3.

| θ | σ_θ (MPa) | | | | | |
|----------|-----------------------|----------|----------|----------|----------|----------|
| | G_{12}/E_1 | 0.2 | 0.4 | 0.6 | 0.8 | 1.0 |
| 0° | | 224.2627 | 182.8274 | 166.3872 | 157.4178 | 151.7376 |
| 10° | | 218.4431 | 185.9280 | 171.6972 | 163.6331 | 158.4209 |
| 20° | | 204.4044 | 194.6964 | 187.6255 | 182.8601 | 179.4900 |
| 30° | | 189.1800 | 207.0080 | 212.5897 | 214.8970 | 215.9955 |
| 40° | | 178.6255 | 218.0841 | 238.2099 | 250.5184 | 258.8396 |
| 50° | | 176.1625 | 221.1711 | 246.0954 | 262.2070 | 273.5433 |
| 60° | | 183.5657 | 212.6519 | 225.2655 | 232.1585 | 236.4414 |
| 70° | | 200.5763 | 196.9797 | 191.3849 | 186.9838 | 183.6381 |
| 80° | | 221.2496 | 183.1520 | 165.6352 | 155.5200 | 148.9160 |
| 90° | | 231.3231 | 177.8302 | 156.6061 | 145.0266 | 137.6935 |
| 100° | | 221.2496 | 183.1520 | 165.6352 | 155.5200 | 148.9160 |
| 110° | | 200.5763 | 196.9797 | 191.3849 | 186.9838 | 183.6381 |
| 120° | | 183.5657 | 212.6519 | 225.2655 | 232.1585 | 236.4414 |
| 130° | | 176.1625 | 221.1711 | 246.0954 | 262.2070 | 273.5433 |
| 140° | | 178.6255 | 218.0841 | 238.2099 | 250.5184 | 258.8396 |
| 150° | | 189.1800 | 207.0080 | 212.5897 | 214.8970 | 215.9955 |
| 160° | | 204.4044 | 194.6964 | 187.6255 | 182.8601 | 179.4900 |
| 170° | | 218.4431 | 185.9280 | 171.6972 | 163.6331 | 158.4209 |
| 180° | | 224.2627 | 182.8274 | 166.3872 | 157.4178 | 151.7376 |

Table 2: The normal stress component σ_θ vs. θ and G_{12}/E_1 for $p=100$ MPa, $E_1=204$ GPa, $E_2/E_1=0.6$, and $\nu_{12}=0.4$.

| θ | σ_θ (MPa) | | | | | |
|----------|-----------------------|----------|----------|----------|----------|----------|
| | E_2/E_1 | 0.2 | 0.4 | 0.6 | 0.8 | 1.0 |
| 0° | | 158.6111 | 163.7024 | 166.3872 | 168.1069 | 169.3123 |
| 10° | | 163.3198 | 168.7368 | 171.6972 | 173.6703 | 175.1163 |
| 20° | | 180.2301 | 184.1946 | 187.6255 | 189.9839 | 191.7611 |
| 30° | | 204.6600 | 210.0193 | 212.5897 | 214.0722 | 214.9998 |
| 40° | | 242.4456 | 240.8558 | 238.2099 | 235.6879 | 233.4666 |
| 50° | | 277.4915 | 257.6026 | 246.0954 | 238.6573 | 233.4666 |
| 60° | | 268.7737 | 237.4698 | 225.2655 | 218.8828 | 214.9998 |
| 70° | | 200.1263 | 192.3720 | 191.3849 | 191.4572 | 191.7611 |
| 80° | | 132.0207 | 155.5936 | 165.6352 | 171.3639 | 175.1163 |
| 90° | | 107.4517 | 142.6084 | 156.6061 | 164.3424 | 169.3123 |
| 100° | | 132.0207 | 155.5936 | 165.6352 | 171.3639 | 175.1163 |
| 110° | | 200.1263 | 192.3720 | 191.3849 | 191.4572 | 191.7611 |
| 120° | | 268.7737 | 237.4698 | 225.2655 | 218.8828 | 214.9998 |
| 130° | | 277.4915 | 257.6026 | 246.0954 | 238.6573 | 233.4666 |
| 140° | | 242.4456 | 240.8558 | 238.2099 | 235.6879 | 233.4666 |
| 150° | | 204.6600 | 210.0193 | 212.5897 | 214.0722 | 214.9998 |
| 160° | | 180.2301 | 184.1946 | 187.6255 | 189.9839 | 191.7611 |
| 170° | | 163.3198 | 168.7368 | 171.6972 | 173.6703 | 175.1163 |
| 180° | | 158.6111 | 163.7024 | 166.3872 | 168.1069 | 169.3123 |

Table 3: The normal stress component σ_θ vs. θ and E_2/E_1 for $p=100$ MPa, $E_1=204$ GPa, $G_{12}/E_1=0.6$, and $\nu_{12}=0.4$.

In Table 3 given $p=100$ MPa, $E_1=204$ GPa, $G_{12}/E_1=0.6$ and $\nu_{12}=0.4$, the variation of the stress σ_θ with respect to the polar angle θ and E_2/E_1 is following:

With $E_2/E_1=0.2, 0.4, 0.6, 0.8$ and 1.0 , the minimum corresponding stresses σ_θ are 107.45 MPa, 142.61 MPa, 156.61 MPa 164.34 MPa, and 169.31 MPa at $\theta=90^\circ$ respectively, and the maximum stresses σ are 277.49 MPa, 257.60 MPa, 246.09 MPa, 238.66 MPa and 233.47 MPa at $\theta=50^\circ$ respectively. Moreover, for $E_2/E_1=1.0$, the minimum stress σ occurred at $\theta=0^\circ$ as well as $\theta=180^\circ$ and the maximum stress σ_θ occurred at $\theta=40^\circ, \theta=130^\circ$ and $\theta=140^\circ$ as well. Also, the stress

concentration factors are $S_c = 2.77, 2.58, 2.46, 2.39,$ and 2.33 respectively, for orthotropic case.

Table 3 shows that within the ranges of the polar angle $0^\circ \leq \theta \leq 30^\circ$, $80^\circ \leq \theta \leq 100^\circ$ and $150^\circ \leq \theta \leq 180^\circ$, the stress σ_θ increases along with the increased values of E_2/E_1 respectively. But, in the ranges of the polar angle $40^\circ \leq \theta \leq 60^\circ$ and $120^\circ \leq \theta \leq 140^\circ$, the stress σ_θ decreases along with the increased values of E_2/E_1 . As for the polar angles $\theta = 70^\circ$ as well as $\theta = 110^\circ$, with the values of $0.2 \leq E_2/E_1 \leq 0.6$, the stress σ_θ decreases along with the increased values of E_2/E_1 , but for the values of $0.6 \leq E_2/E_1 \leq 1.0$, the stress σ_θ increases along with the increased values of E_2/E_1 . A summary of Table 3 is shown in Figure 4.

Conclusions

The effect of lamina material properties on the stress distribution of the orthotropic composite plate containing a circular cutout under hydrostatic tension is presented.

First of all, given fixed material parameters for $E_1 = 204$ GPa, $E_2/E_1 = 0.6$, and $G_{12}/E_1 = 0.4$ Table 1 indicates that for the values of $0.2 \leq \nu_{12} \leq 0.6$, within the ranges of the polar angle $0^\circ \leq \theta \leq 50^\circ$ and $90^\circ \leq \theta \leq 130^\circ$, the stress σ increases with respect to the increased values of the polar angle θ , but within the ranges of the polar angle $50^\circ \leq \theta \leq 90^\circ$ and $130^\circ \leq \theta \leq 180^\circ$ the stress σ_θ decreases with the increased values of the polar angle θ .

Second, given fixed material parameters for $E_1 = 204$ GPa, $E_2/E_1 = 0.6$, and $\nu_{12} = 0.4$ Table 2 indicates that for the value of $G_{12}/E_1 = 0.2$, within the ranges of the polar angle $0^\circ \leq \theta \leq 50^\circ$ and $90^\circ \leq \theta \leq 130^\circ$, the stress σ_θ decreases with respect to the increased values of the polar angle θ , but within the ranges of the polar angle $50^\circ \leq \theta \leq 90^\circ$ and $130^\circ \leq \theta \leq 180^\circ$ the stress σ_θ increases along with the increased values of the polar angle θ .

As for the values of $0.4 \leq G_{12}/E_1 \leq 1.0$, within the ranges of the polar angle $0^\circ \leq \theta \leq 50^\circ$ and $90^\circ \leq \theta \leq 130^\circ$, the stress σ_θ increases with the increased values of polar angle θ , but within the ranges of the polar angle $50^\circ \leq \theta \leq 90^\circ$ and $130^\circ \leq \theta \leq 180^\circ$ the stress σ_θ decreases with respect to the increased values of the polar angle θ .

Third, given fixed material parameters for $E_1 = 204$ GPa, $G_{12}/E_1 = 0.6$, and $\nu_{12} = 0.4$ Table 3 indicates that for the values of $0.2 \leq E_2/E_1 \leq 1.0$, within the ranges of the polar angle $0^\circ \leq \theta \leq 50^\circ$ and $90^\circ \leq \theta \leq 130^\circ$ the stress σ_θ increases with respect to the increased values of the polar angle θ , but within the ranges of the polar angle $50^\circ \leq \theta \leq 90^\circ$ and $130^\circ \leq \theta \leq 180^\circ$ the stress σ_θ decreases along with the increased values of the polar angle θ .

Fourth, it is well known that for fiber reinforced composite laminated plates, the Poisson's ratio can be negative values. However, this case is not discussed/analyzed in the present study; such investigation will be included in the future research works.

The results obtained from this dimensionless analysis provide a set of general design guidelines for structural laminates with high precision requirements in the engineering applications.

References

1. Niu, Michael CY (1992) Composite Airframe Structures. Conmil Press Ltd, USA.
2. Sadd MH (2009) Elasticity, 2 edn. Elsevier Inc., USA.
3. Kuwamura H (2010) Splitting of Wood by Pressure in a Circular Hole: Study on Steel-Framed Timber Structures part 5. Journal of Structural and Construction Engineering, 75: 175-184.
4. Selivanov MF (2010) Influence of the Viscoelastic Properties of a Composite on the Stress Distribution around an Elliptic Hole in a Plate. International Applied Mechanics, 46: 799-805.
5. Kumar RR, Rao GV, Suresh KS (1994) Normal Stress Distribution adjacent to Optimum Holes in Composite Plates. Composite Structures, 29: 393-398.
6. Giare GS, Shabahang R (1989) Reduction of Stress Concentration around the Hole in an Isotropic Plate Using Composite Materials. Engineering Fracture Mechanics 32: 757-766.
7. Tsai KH, Hwan CL, Lin MJ, Huang YS (2012) Finite element based point stress criterion for predicting the notched strengths of composite plates. Journal of Mechanics, 28: 401-406.
8. Sih GC, Paris PC, Irwin GR (1965) On Crack in Rectilinearly Anisotropic Bodies. International Journal of Fracture, 1: 189-203.
9. Lekhnitskii SG (1968). Anisotropic Plates. USA.
10. Tan SC (1994) Stress Concentrations in Laminated Composites. USA.